Appendix II. - Statistical Size Effect

The decrease of observed nominal strength with the size of specimen has usually been explained statistically. The strength value is assumed to vary randomly throughout the material, and so in a specimen of a larger size the chance of encountering a smaller minimum strength is higher. The size effect law obtained in this manner then depends on the assumed statistics of strength distribution, or the statistics of the size of microscopic flaws which cause the local strength reduction. This classical, statistical approach has, however, several significant differences from the present size effect law.

One difference is due to the fact that for volumes larger than a certain characteristic volume the chance of encountering a still smaller strength (or a still larger flaw) does not increase. In such a case the statistically based size effect reaches an asymptotic value at a certain specimen size, and does not decrease any more for larger specimen sizes. On the other hand, the present size effect law continues decreasing.

Another, more significant difference is due to the fact that the amount of energy released from the uncracked part of the structure into the fracturing region is very important. Thus, the same specimen supported rigidly or elastically would have the same strength according to the statistical theory, while according to the present energy-based size effect law the elastically supported specimen can have a much smaller strength. Obviously, a similar phenomenon arises from the geometry of the structure, in the various possible manners the geometry of the uncracked regions of the structure can significantly affect the failure.

Some statistical size effect surely exists, but it may be much less significant than the present energy-based size effect. At least for concrete it seems at the present that all the known size effect can be adequately explained by the energy-based size effect law, although statistically-based theories have previously been offered to provide an explanation.

SEA ICE INDENTATION ACCOUNTING FOR STRAIN-RATE VARIATION

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ABSTRACT: Global and local indentation pressures in the creeping mode of sea ice deformation are obtained, accounting for the spatial considered; the upper bound and strain path methods of analysis are postulated velocity fields required in the analysis are calibrated with power-law creep model and by the multi-axial extension of a new uniaxial model which accounts for both hardening and softening behavior. Results are compared with previously published indentation formulas.

INTRODUCTION

Two levels of ice loading are typically considered in the design of govern the production platforms for the Arctic. Global ice pressures foundation design, while local pressures are likely to dictate wall thicknesses and local framing, and may well govern structural cost. Most of the emphasis on ice force research has been on predicting global feasibility to preliminary and detailed design, has the importance of times the average global pressure. It is widely recognized that uncersome cases design loads may be overestimated by an order of magnitude. Uncertainties in existing for load prediction models in use today and that in

Uncertainties in existing to overestimated by an order of magnitude. sources: (1) incomplete modeling of the thermomechanical behavior of sea ice, (ii) use of semi-empirical formulations, calibrated without to realistically model the contact forces at the ice-structure interface ness of the environmental forces driving the ice features. Both approthe finite and boundary element methods of analysis can be used to study material and interface behavior.

This paper employs two approximate methods of analysis, the upper bound and strain path methods, to study the problem of sea ice indentation in the creeping mode of deformation, accounting for the spatial variation of strain-rates. This is a problem of concern for artificial islands in the Arctic nearshore region, where "break-out" and/or steady indentation conditions occurring in the winter form a basis for select-

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ing design ice loads. The key difference in the two analyses is that point stresses within the continuum can be obtained with the strain path method. As a result, local stresses at the ice-structure interface can be estimated, unlike the upper bound method which only yields the global pressure. However, both methods rely on an adequate specification of the velocity field in the ice sheet. This is obtained through a combination of theoretical modeling based on fluid mechanics and field ice movement survey data from an artificial island in the Beaufort Sea. In particular, two theoretical kinematic models are considered: one resulting from the superposition of a point source and a uniform flow (Kinematic Model A) that has been studied previously (3,9); and the other from the superposition of a doublet and a uniform flow (Kinematic

An important aspect of the analysis is the specification of the mechanical behavior of sea ice. In order to provide continuity with previous work, the isotropic, incompressible three-dimensional extension of the uniaxial power-law creep model is studied. The predicted global ice pressures are compared with those from previously published formulas (1,8). Finally, a new uniaxial law that models the stress-strain-strainrate behavior of sea ice, including its strain-softening behavior, is presented. The strength-strainrate relationship derived from this new model is used to predict global ice pressures during indentation.

BOUND METHOD VERSUS STRAIN PATH METHOD

An upper bound (conservative) solution to a continuum mechanics problem may be derived by relaxing the statical field equations and boundary conditions, and using velocity fields that satisfy the kinematic constraints for the problem. Applying the principle of virtual work and Drucker's convexity criterion, the upper bound estimate of the global load for incompressible materials may be obtained with (4):

$$\begin{cases}
S_{1j}^* \stackrel{\epsilon^*}{ij} dV \ge \int T_1 U_1^* dS + \int T_1 U_1 dS \\
V S_T
\end{cases} \qquad S_T$$
(1)

where $S_{1,1}^*$, $\hat{c}_{1,1}^*$ and U_1^* are the deviatoric stresses, strain-rates and velocities obtained from an assumed kinematically admissible velocity field. T_1 and U_1 are the actual surface tractions and velocities.

The upper bound method does not make use of the field equilibrium equations. As such, point stresses in the continuum are unknown. Hill (5) has suggested an approximate method by which octahedral (hydrostatic) stresses can be derived from deviatoric stress gradients using the equilibrium equation. This idea was developed and first applied to deep penetration problems in soil mechanics by Baligh (2), who called it the strain path method.

In applying the strain path method to the ice indentation problem, the major assumption is that the strain and strain-rate field can be obtained from the kinematic conditions with no reference to constitutive relations, equilibrium equations, or statical boundary conditions. This is an approximation and hence the derived stress field is approximate in general. However, the method is computationally very attractive when compared with a detailed finite element analysis. In addition, the method provides valuable insights to the indentation problem which is difficult to obtain from a purely numerical approach.

KINEMATIC MODELING OF ICE SHEET

Theoretical Kinematic Models.-- Kinematic Model A is shown in Fig. 1. The streamline passing through the stagnation point defines the bluff-body, i.e., the region where the oncoming sheet of ice cannot enter. The streamfunction, ψ , and flow velocities, U and U_{θ} , follow from the theory of fluid mechanics and are given by:

$$\psi = -U_{O} r \sin \theta - U_{O} a \theta \tag{2}$$

$$U_{r} = \frac{-1}{r} \frac{\partial \psi}{\partial \theta} = U_{o} \cos \theta + U_{o} \frac{a}{r}$$
(3)

$$U_{\theta} = \frac{\partial \psi}{\partial \mathbf{r}} = -U_{o} \sin \theta \tag{4}$$

where U_0 is the uniform far field velocity. The bluff-body is described by $r=a(\pi-\theta)/\sin\theta$ with the half-width of the body at $r=\infty$ equal to πa . This model assumes that the normal velocity at the ice/bluff-body interface is zero. Even if this is valid, the tangential contact between the moving ice sheet and the half-body could in general be either friction-free or possess finite frictional forces. This imposes a statical best dary condition with which the chosen velocity field may or may not be consistent.

Kinematic Model B is shown in Fig. 2. The bluff-body in this case is a circle of radius, a. This represents flow past a cylindrical indenter with contact at all points on the circumference. The streamfunction and flow velocities are given by:

$$\psi = - U_{\text{or}} \sin \theta + U_{\text{o}} \frac{a^2}{r} \sin \theta \tag{5}$$

$$U_{\mathbf{r}} = U_{0} \left[1 - \frac{\mathbf{a}^{2}}{\mathbf{r}^{2}} \right] \cos \theta \tag{6}$$

$$U_{\theta} = -U_{0} \left[1 + \frac{a^{2}}{r^{2}}\right] \sin \theta$$

(7)

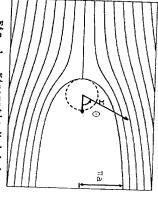


Fig. 1.--Kinematic Model A

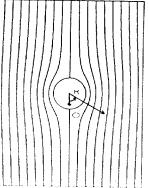


Fig. 2.--Kinematic Model B

Calibration with Field Ice Movement Data. -- The theoretical velocity fields are calibrated with field ice movement data from an artifical island in the Beaufort Sea, obtained over a period of seven weeks during beak winter ice formation. The surveys were carried out at the 39 stations at least once per day, although during high movement events the

approximately equal to the average of the predicted velocities over the observation period; and (c) the measured direction of movement must equal the predicted bearing. flow conditions; (b) the average movement rate measured should be the start and end of the observation period should be equal under steady following three criteria: surveys were made every 12 hours. (a) streamfunction values for a particle at The calibration is based on the

larger, can be explained with arguments similar to that for velocities. Model B and larger for Model A. The discrepancies, where they are quately modeled by both velocity fields, (b) Model B is in general 5% to 10% if the above exceptions are taken into account. The errors in better than Model A. Typical errors in the velocity are on the order of macrocracking: (a) separate and large regions of the continuum are aderal geometry, e.g., choice of diameter D. In the cases affected by are significantly in error due to an inadequate modeling of the structumodels. Some stations near the rubble pile surrounding the structure bearing predictions are on the order of 5 to 10 degrees for Kinematic generally less than 1% in all cases except the case with macrocrack when there is no macrocrack formation. For Model B, the errors are formation. The errors in velocity predictions are much higher for both For Model A the errors vary from less than 1% to as much as 25%, even Nevertheless, both kinematic models predict relatively accurate the bluff-body cannot model field conditions prior to macrocracking. downstream side. Thus Kinematic Model A which considers no flow within magnitude as on the upstream side prior to macrocracking, which was streamfunction values over their respective domains of applicability. observed to occur in a direction perpendicular to the flow on the stream and directly at the back of the indenter are of the same order of A careful analysis of the data shows that the ice movements down-

macrocracking occurs for velocities less than 1 ft/hr $(85 \times 10^{-6} \text{ m s}^{-1})$. upstream flow field; and $\langle \mathcal{L}_i \rangle$ the transition from creeping behavior to side; (c) even after macrocracking, Model B accurately models the observed macrocracking if tensile stresses develop on the downstream description of the measured flow field and can be used to explain the field data considered here; (b) Kinematic Model B provides a good In summary: (a) Kinematic Model A does not adequately model the

ICE PRESSURES FOR POWER-LAW CREEP MODEL

uniaxial power-law creep leads to the following constitutive model: The isotropic, incompressible three-dimensional extension of the

$$S_{1j} = \frac{2}{3} \left(\begin{array}{c} o \\ e \\ \end{array} \right)^{\frac{1}{N}} \stackrel{\text{Leep reads to the following constitutive r}}{e}$$
(8)

where s_{ij} and $\hat{\epsilon}_{ij}$ are the deviatoric stress and strain-rate tensors, respectively. The effective strain-rate is defined as:

$$\dot{\epsilon} = \left[\frac{2}{3} \sum_{i,j} \frac{\epsilon}{i,j}\right]^{1/2} \tag{9}$$

N is the power-law exponent; $\sigma_{\rm O}$ and $\epsilon_{\rm O}$ are material constants derivable from unlaxial testing of ice. The results presented in this paper are based on the sea ice data of Wang (10).

are: (a) compute the strain-rate field by differentiating the veloci-The basic steps for evaluating stresses using the strain path method

> with the bound method and/or by integration of the local stress field. from the total stress field, while global pressures may be estimated Then, the local stresses at the ice-structure interface can be obtained octahedral stresses will in general be path dependent), and (d) estimate dral stresses by spatially integrating the equilibrium equations (the toric stress field using the constitutive equations, (c) obtain octahethe total stresses by summing the deviatoric and octahedral stresses. ties with respect to the spatial coordinates, (b) evaluate the devia-

orthogonal direction with the stress gradients in that direction. magnitude of fictitous body forces required for equilibrium in the satisfying statical boundary conditions. Another way of making the along different (i.e., orthogonal) paths and by noting the error in assessed by comparing the octahedral stresses obtained by integration former comparison is to integrate along one path and to compare the The degree of approximation in the estimated stress field can be

are presented below. bound and strain path methods of analysis to the chosen kinematic model: Ice Pressure Prediction .-- Results from the application of the upper

may not be satisfied by the model. In addition, the following comments can be made: satisfied. In spite of this, the statical interface boundary condition 1. Kinematic Model A: The octahedral stress field for this kinematic model is path independent and as such equilibrium is exactly

(i) the maximum strain-rate occurs at the ice/bluff-body

interface at r=a, and is equal to U_0/a , (11) the stress field decays as $r^{-2/N}$, which for N=4 is $1/\sqrt{r}$,

(iii) the stress field is axisymmetric, and

(iv) the octahedral stress is zero for a viscoelastic material (N=1).

assuming a frictionless ice/bluff-body interface. This is similar to the case studied by Bruen and Vivatrat (3), and Eq. (1) reduces to: The global pressure can be estimated by the upper bound method,

$$\frac{P}{2at} \leq \frac{N}{\sqrt{3}} \int_{0}^{\pi} \left(\frac{\sin \theta}{\theta}\right)^{2/N} d\theta \left[\frac{2}{\sqrt{3}} \begin{array}{ccc} \sigma & U & \frac{1}{2} \\ \vdots & \sigma & \frac{1}{2} \end{array}\right]^{N}$$
(9)

integration of the local stresses around the bluff-body, i.e., A second approach to estimating the global pressure involves

$$P = t \int (\sigma_{rr} \cos \theta r d\theta + \sigma \theta \theta \sin \theta dr)$$
 (10)

motion. The implications of this finding are more fully discussed for upper bound method, that method also predicts zero pressure. However, the bound theorems do not apply for the case of friction with relative Kinematic Model B. Strain path method at the ice/bluff-body interface are included in the the upper bound method. If the frictional forces predicted by the Using the equation for the bluff-body, Eq. (10) shows that P=0, unlike

- model are path dependent, although for N=1 and N=3 the fictitious body In addition, the following comments can be made: tical interface boundary conditions will in general not be satisfied. force is zero and equilibrium is satisfied exactly. However, the sta-2. Kinematic Model B: The octahedral stresses for this kinematic
- (1) the maximum strain-rate occurs at the ice-structure interface, rma, and is equal to $2\mathrm{U}_\mathrm{O}/\mathrm{a}$ or $4\mathrm{U}_\mathrm{O}/\mathrm{D}$,

(ii) the stress field decays as r-3/N (not r-2/N as predicted by Kinematic Model A), which for N=4 is r-3/4,

(iii) the stress field is not axisymmetric, and

(iv) the octahedral stress is zero for a viscoelastic material. The radial stresses downstream of the indenter are tensile and equal in magnitude to the upstream compressive stresses, consistent with the material law. The tangential stresses behave similarly for typical values of N, although their magnitudes could be less than half of the radial stresses. These are principal stresses at 0=0. As such, it is reasonable to expect a macrocrack formation on the downstream side of the indenter. This is indeed borne out by the field ice movement data.

The global pressure can be estimated using the bound method, assuming either a frictionless interface or an interface with frictional stresses predicted by the strain path method. (The method does not strictly apply for the latter case.) The respective expressions, derived from Eq. (1) for D=2a, are:

$$\frac{\text{No friction}}{\text{Dt}} \leq \frac{4\pi}{\sqrt{3}} \frac{\text{N}}{\text{N+3}} \left[\frac{4}{\sqrt{3}} \frac{\text{o}}{\hat{\epsilon}_{0}} \frac{2\text{U}}{\text{D}} \right] \frac{1}{\text{N}}$$

$$\frac{\text{With friction}}{\text{Dt}} \leq \sqrt{3\pi} \frac{\text{N-1}}{\text{N+3}} \left[\frac{4}{\sqrt{3}} \frac{\text{o}}{\hat{\epsilon}_{0}} \frac{2\text{U}}{\text{D}} \right] \frac{1}{\text{N}}$$

$$(11)$$

The ratio of Eq. (11) to Eq. (12) is 4N/3(N-1), which varies between 1.8 and 2.2 for $2.5 \le N \le 4$. Intuitively, interface friction should increase indentation pressures. However, both kinematic models studied here predict a significant decrease in pressure. This is because they are derived from considerations for ideal, non-viscous fluids and as such do not correctly model interface conditions. Even if more exact velocity fields can be postulated theoretically, the available field data does not provide adequate resolution of the ice movements in the immediate vicinity of the structure to calibrate the kinematic models.

Integration of the local stresses around the bluff-body yields another estimate of global pressure accounting for the frictional stresses of the strain path method:

$$\frac{P}{Dt} = \frac{\pi}{2\sqrt{3}} (N-1) \left[\frac{4}{\sqrt{3}} \frac{\sigma}{\dot{\epsilon}} \frac{2U}{D} \right]^{\frac{1}{N}}$$
 (13)

Both Eqs. (12) and (13) predict zero pressure for a viscoelastic material. Furthermore, the ratio of the upper bound method to Eq. (13) is 6/(N+3), which varies between 0.86 and 1.09 for 2.5

For the artificial island considered in this paper the maximum strain-rate just prior to macrocracking is on the order of 10-6 s-1 or less. The peak local tangential stresses are on the order of 80 psi (0.55 MPa), the peak local radial stresses are on the order of 320 psi (2.2 MPa). The radial stresses are compressive upstream of the indenter and tensile on the downstream side, and are distributed in a cosinusoidal fashion. The typical order of magnitude value for the global pressure obtained with Eq. (13) is about 350 psi (2.4 MPa). A key inference here is that the local and global pressures are on the same order of magnitude. If instead Eq. (11) is used to estimate the global pressure, the peak local pressure becomes approximately half the global pressure.

Table 1.--Comparison of Average Global Pressures for Power-Law Creep with N=4 and Average Strain-Rate of $\rm U_{\odot}/2D$

Model B	Model A D = 2πa		Ponter et al. (1983) Plane Stress		API Plane Stress		Model Constraints
7.22	7.60	3.22	1.85	3.64	3.13	4.12	φ

Comparison with Other Ice Pressure Formulas. -- Average global pressures during sea ice indentation can be estimated using any one of the many predictive models available in the literature. In this study the global pressures predicted by Kinematic Models A and B, neglecting interface frictional stresses, are compared with the models of API (1) and Ponter et al. (8). The general form of all these models is given by:

 $\frac{P}{Dt} = \phi \circ (\hat{\epsilon})$ (14)

where ϕ is a constant depending in general on N, and $\sigma(\epsilon_a)$ is the uniaxial strength of ice evaluated at some average strain-rate, $\epsilon_a=U_0/(\phi\Psi D)$ with Ψ being a second constant. In order to compare the various formulations, $\phi\Psi$ is assumed equal to two as suggested by API (1)

apply for a flat indenter with ice pressures being allowed to develop only on the upstream side. For the API model, the sea ice is assumed only on the upstream side. and the comparison can therefore be based on the parameter $\boldsymbol{\phi}$. model, a correction factor of 1.1 is applied to ϕ to make results concolumnar and the contact factor is set to one. In Ponter et al.'s creep model with N=4 is given in Table 1. At first glance the numbers Model B is based on a circular indenter with compressive and tensile stresses on the upstream and downstream sides respectively. This is away $(r_{+\infty})$ from the tip of the bluff-body with the region in between sistent with N=4. In Kinematic Model A, the problem geometry (Fig. 1) $\,$ important differences among the models. The first two formulations seem highly scattered, varying from 1.85 to 7.60. However, there are consisting of inert ice, while in the latter case the indenter is 42% of that for D=2a. In the former case the indenter is located far is subjective. If the indenter diameter is chosen as $2\pi a$, φ is about is different from the other models and the choice of indenter diameter more representative of actual field conditions prior to breakout. indicate that both assumptions may be unrealistic. The ϕ factor for the indenter. Field data on deformation patterns considered here located at the tip of the bluff-body with the inert region downstream The values of ϕ predicted by the four formulations for a power-law

If the API and Ponter et al. models are extended to account for downstream tensile stresses, the ϕ factors would probably be twice as much since for the problem and material model considered (a) tensile and compressive strengths are equal, and (b) stress levels are equal but opposite in sign on the upstream and downstream sides. Then, φ for the

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API model would vary between 6.3 and 8.2 while for Ponter et al.'s it would vary between 3.7 and 7.3. For the artificial island considered earlier and other typical artificial islands with D/t>20, the behavior is closer to a plane stress condition. Both Model B and the API model tend to predict similar global pressures under this scenario.

ICE PRESSURES FOR NEW UNIAXIAL MODEL

The unlaxial behavior of sea ice has often been idealized with the power-law creep formulation. This model relates the strength of ice to the strain-rate, and as such is an incomplete description of the stress-strain-strainrate characteristics of the material. Several investigators have proposed more complete unlaxial models, some empirical in nature, that seem to be able to reproduce the post-peak decrease of stress in ice under constant strain-rate conditions. However, most of these models have not been fit to any specific data on sea ice. In addition, since they represent behavior only under constant strain-rate conditions, it is difficult to extend the models to other loading conditions such as constant stress (creep) and stress-rates.

A phenomenological approach based on simple thermorheological models is used here to develop a new uniaxial model, which is then calibrated with the sea ice data of Wang (10). This model applies equally well for constant stress-rate and creep conditions. In addition, under constant strain-rate conditions the strain at peak stress is a function of the strain-rate. The strength-strainrate relationship derived from the new uniaxial model is extended for multi-axial stress states and then applied to obtain indentation pressures.

New Uniaxial Constitutive Model. The new uniaxial constitutive model is based on the concept that the strength of an ice specimen is affected simultaneously by work or strain hardening and work softening or recovery. The latter phenomenon may occur due to recrystallization or micro-voids formation. This concept was used by Orowan (7) for examining steady state creep, and may be expressed as:

$$\frac{d\sigma}{dt} = \frac{\partial\sigma}{\partial\epsilon} \cdot \frac{\partial\sigma}{\partial\epsilon}$$
 (15)

where $h=\partial\sigma/\partial\epsilon$ is the coefficient of work hardening and $r=-\partial\sigma/\partial t$ is the rate of recovery. The parameter h is generally modelled as (6):

$$h = A \frac{1}{\varepsilon} 1/N \exp (-M\varepsilon)$$
 (16)

where Q is the activation energy and R is the universal gas constant; A, N and M are the parameters of the equation. Due to a lack of general models for work softening, it is assumed here that the form is similar to that for work hardening:

$$r \approx B \epsilon L/K \exp (-L\epsilon)^{\frac{1}{2}}$$
 (17)

with B, K and L being the parameters of the equation. Substituţing Eqs. (16) and (17) into Eq. (15) yields after integration (assuming ϵ and τ are constant with $\epsilon = \epsilon t$ and $\sigma = 0$ at t = 0):

$$\sigma = \left[\frac{A}{M} \frac{\hat{\epsilon}^{1/N} \{1-\exp(-M\epsilon)\}}{1-\exp(-M\epsilon)}\right] - \frac{B}{L} \frac{\hat{\epsilon}^{1/K-1} \{1-\exp(-L\epsilon)\}}{1-\exp(-L\epsilon)}$$
(18)

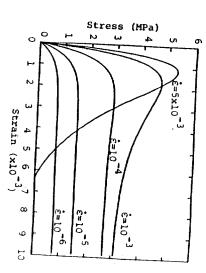


Fig. 3.--Stress-Strain Curves for Constant Strain-Rate Using New Model

The parameters of Eq. (18) obtained from Wang's (10) experimental data are: A=114025 MPa s¹/3, B=217408 MPa s²/3, M=1411.2, L=430, N=3 and K=0.6. The stress-strain plots for the new model are shown in Fig. 3.

The two plots of the stress-strain behavior for $\epsilon=10^{-3}$ s⁻¹ shown in Fig. 3. Ref. 10 are vastly different, one shows strain softening behavior with resultant residual strength, while in the other the stress reduces to zero very sharply for strains exceeding the peak stress. The new model (Fig. 3) reflects this behavior for strain-rates of 10^{-2} - 10^{-3} s⁻¹, rate in this region.

Ice Pressure Prediction. The uniaxial model developed here relates the stress to strain and strain-rate. The strength-strainrate relation-under the assumptions of isotropy and incompressibility made earlier for the power-law creep model. These assumptions are reasonable for the strain-rates of interest in this paper. Ice pressures obtained with model can then be compared with the results for the power-law creep and temperature is currently under development.

The strain-rate is currently under development.

The strength-strainrate relationship for the new uniaxial model is plotted in Fig. 4 and compared with the power-law creep model. Notice that the new model predicts lower strength at strain-rates less than effective power-law exponent is an increasing function of continuous that the

The new constitutive model can be viewed as the superposition of three power-law models of the type considered in the previous section and as such the local pressures will be distributed similarly. A considered here, based on Kinematic Model B and Eq. (13), is about 250 power-law creep model. The 30% reduction in pressure is significant, pressure in the latter case is about 480 psi (3.3 Mpa).

Log Strength

(MPa)

Fig. 4.--Comparison of Uniaxial Strength-Strainrate Relations

CONCLUSIONS

This study of global and local indentation pressures in the creeping mode of sea ice deformation, accounting for the spatial variation of strain-rates, leads to the following specific conclusions:

 Kinematic Model B better models the ice movement survey data obtained from an artificial island in the Beaufort Sea than Kinematic Model A.

- 2. In the creeping mode of ice deformation, the local ice pressures are of the same order of magnitude as or lower than the global pressures. Even if the global pressures are reduced, e.g., by a factor of three, to account for scale (fracturing) effects, the local pressures based on the strain path method will only be 1.5 (and not three) times the upper bound global pressure neglecting interface friction.
- the API model (1) predict similar global pressures. For a typical artificial island just prior to break-out with ice movements of less than 1 ft/hr (85x10⁻⁶ m s⁻¹), Model B predicts a pressure (neglecting interface friction) of approximately 530 psi (3.7 MPa).
- 4. The constitutive law based on the new uniaxial model predicts global pressures that are lower than that from the power-law creep model. For a typical artificial island, this reduction can be as much as 10-30 percent.

A key finding of this work is that for the rate-dependent material models describing sea ice behavior, interface adfreeze and friction stresses may significantly influence both local and global ice pressures. This has major economic consequences for platform design. Incorporation of these "non-conservative" stresses within the bound method may yield more accurate global ice pressures, but the solutions will not necessarily be upper bounds. More exact estimates of both local and global ice pressures using the strain path method may be

STRAIN RATE VARIATION

obtained by postulating kinematic models that more correctly model the interface conditions. However, currently available field data does not provide adequate resolution of the ice movements in the immediate vicinity of the structure to calibrate such models. In conclusion, it appears that the development of numerical models based on the finite and boundary element methods of analysis is necessary for more realistically pressures are of interest.

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